

# **Configurational Equilibrium of Cracks Affected by Surface Stress and Crack-Tip Point Load**

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# Configurational Equilibrium

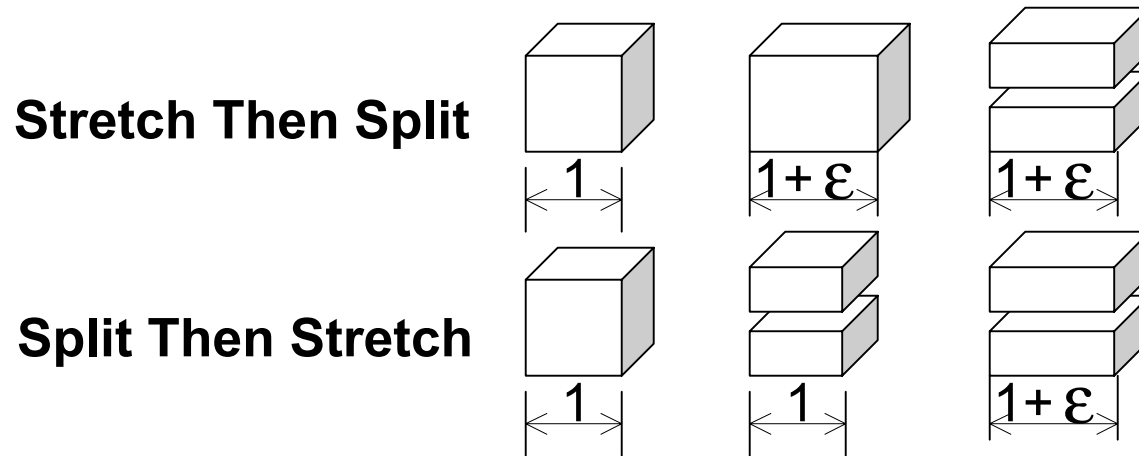
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$$\frac{\partial(\text{Potential Energy})}{\partial(\text{Configuration})} = 0$$

**Effect of Surface Stress and Crack-Tip Load**

# Surface Stress



**Stretch unit cube along x, then split**

$$W(\epsilon) + 2\Gamma(\epsilon) = \text{Work A}$$

**Split unit cube, then stretch along x**

$$2\Gamma(0) + (W(\epsilon) + 2\Sigma\epsilon) = \text{Work B}$$

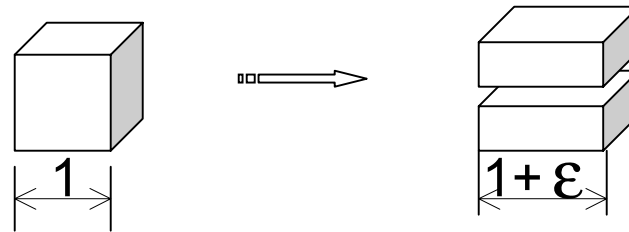
$$\text{Work A} = \text{Work B}$$

$$\Sigma = \partial\Gamma / \partial\epsilon$$

# Referential and Spatial Forms

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$\Gamma(\varepsilon)$ : **Surface Energy per Unit Referential Area**

$\gamma(\varepsilon)$ : **Surface Energy per Unit Spatial Area**

$$\gamma(\varepsilon) = \frac{1}{1 + \varepsilon} \Gamma(\varepsilon) = (1 - \varepsilon) \Gamma(\varepsilon), \quad \Sigma = \frac{\partial \Gamma}{\partial \varepsilon} = \gamma(\varepsilon) + \frac{\partial \gamma}{\partial \varepsilon}$$

**Linear Density:**  $\Gamma(\varepsilon) = \Gamma_o + \Sigma_o \varepsilon, \quad \gamma(\varepsilon) = \Gamma_o + (\Sigma_o - \Gamma_o) \varepsilon$

$\Gamma_o$ : **Surface Tension Coefficient**

$\Sigma_o$ : **Surface Stress Coefficient**

# Surface Stress Coefficient

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**Behaves like a pre-stressed membrane that is perfectly fitted on the bounding surface of a bulk material body.**

**Interacts with the deformation of the bulk material through the curvature of the surface.**

**Becomes increasingly important at reduced scales.**

**Effect on linear and curvilinear cracks.**

# Configurational Energy

Griffith Crack (1921 & 1924)

**Configurational Energy:**

$$\Pi = \Pi_0 + \Delta\Pi$$

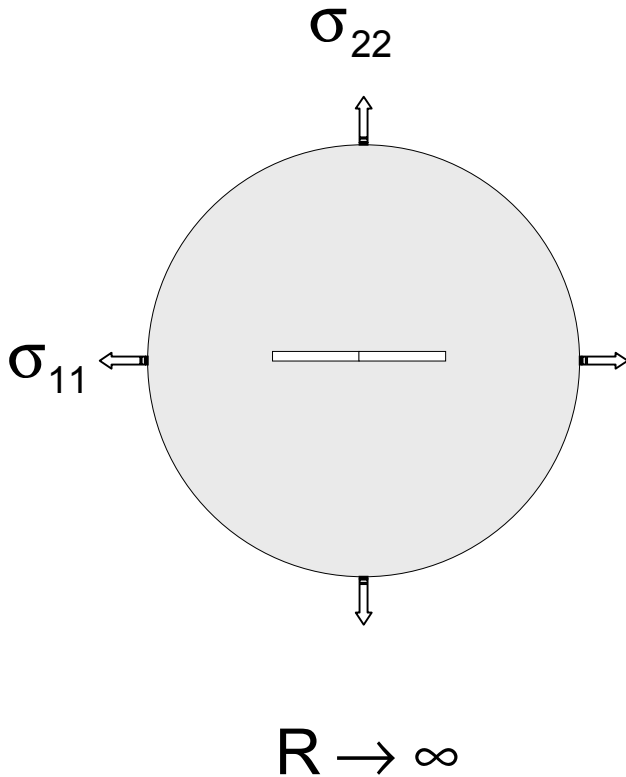
**Stress Intensity Factor:**

$$K_1 = \sigma_{22} \sqrt{\pi a}$$

**For Configurational Equilibrium:**

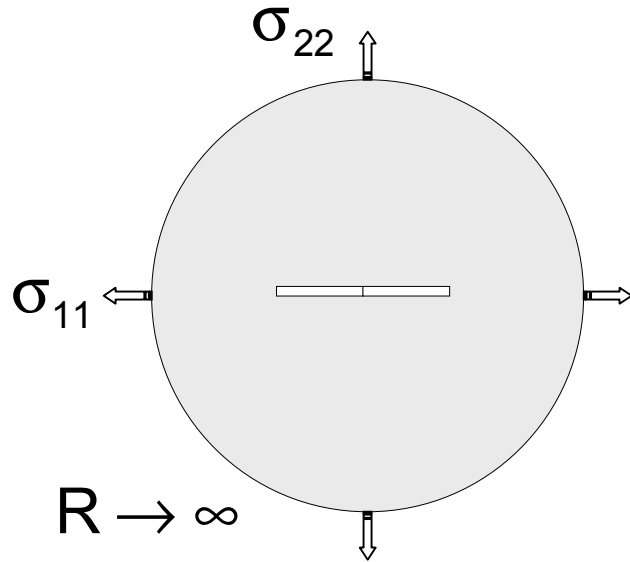
$$\Delta\Pi = -\frac{\kappa + 1}{8\mu} \pi a^2 \sigma_{22}^2 + 4\Gamma_0 a$$

$$\frac{\partial \Delta\Pi}{2\partial a} = 0 \Rightarrow K_1^2 = K_{1C}^2 \equiv \frac{16\mu\Gamma_0}{\kappa + 1}$$



# Configurational Energy

Griffith Crack with Surface Stress  $\Sigma_0$



**Configurational Energy:**

$$\Pi = \Pi_0 + \Delta\Pi$$

**Stress Intensity Factor:**

$$K_1 = \sigma_{22} \sqrt{\pi a} \quad (\text{same})$$

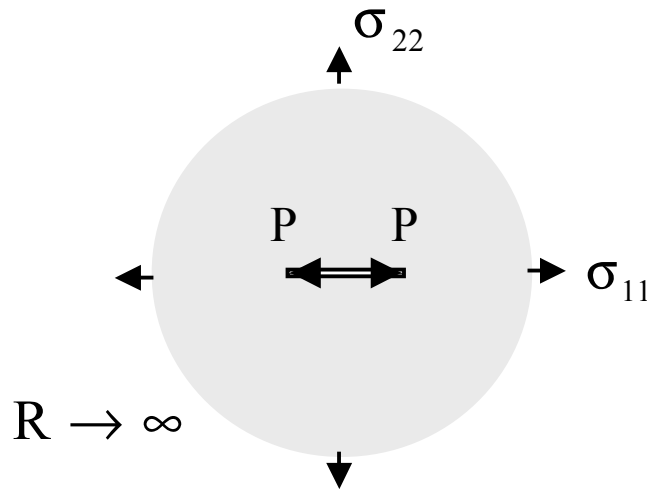
**For Configurational Equilibrium:**

$$\Delta\Pi = -\frac{\kappa+1}{8\mu} \pi a^2 \sigma_{22}^2 + 4\Gamma_0 a - \frac{\kappa+1}{8\mu} 4a\Sigma_0 (\sigma_{22} - \sigma_{11})$$

$$\frac{\partial \Delta\Pi}{2\partial a} = 0 \quad \Rightarrow \quad K_1^2 + \frac{2\Sigma_0}{\sqrt{\pi a}} \left( 1 - \frac{\sigma_{11}}{\sigma_{22}} \right) K_1 - K_{1C}^2 = 0$$

# Configurational Energy

Griffith Crack with Surface Stress  $\Sigma_0$  and Crack-Tip Point Load P



**Configurational Energy:**

$$\Pi = \Pi_0 + \Delta\Pi$$

**Stress Intensity Factor:**

$$K_1 = \sigma_{22} \sqrt{\pi a} \quad (\text{still the same})$$

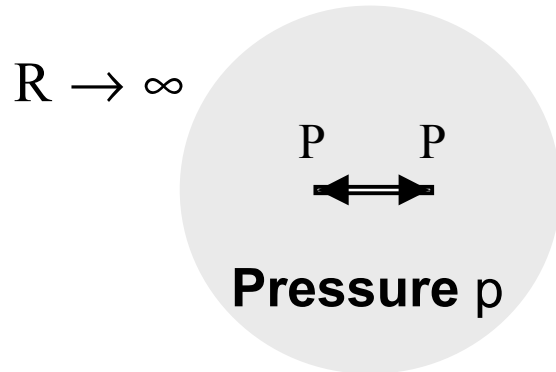
**For Configurational Equilibrium:**

$$\Delta\Pi = -\frac{\kappa + 1}{8\mu} \pi a^2 \sigma_{22}^2 + (4\Gamma_0 - 2P)a - \frac{\kappa + 1}{8\mu} (4\Sigma_0 - 2P)(\sigma_{22} - \sigma_{11})a$$

$$\frac{\partial \Delta\Pi}{2\partial a} = 0 \quad \Rightarrow \quad K_1^2 + \frac{2\Sigma_0 - P}{\sqrt{\pi a}} \left(1 - \frac{\sigma_{11}}{\sigma_{22}}\right) K_1 - \left(1 - \frac{P}{2\Gamma_0}\right) K_{1C}^2 = 0$$

# Configurational Energy

Surface Stress, Internal Pressure  $p$ , and Crack-Tip Load  $P$



**Configurational Energy:**

$$\Pi = \Pi_0 + \Delta\Pi$$

**Stress Intensity Factor:**

$$K_1 = p\sqrt{\pi a}$$

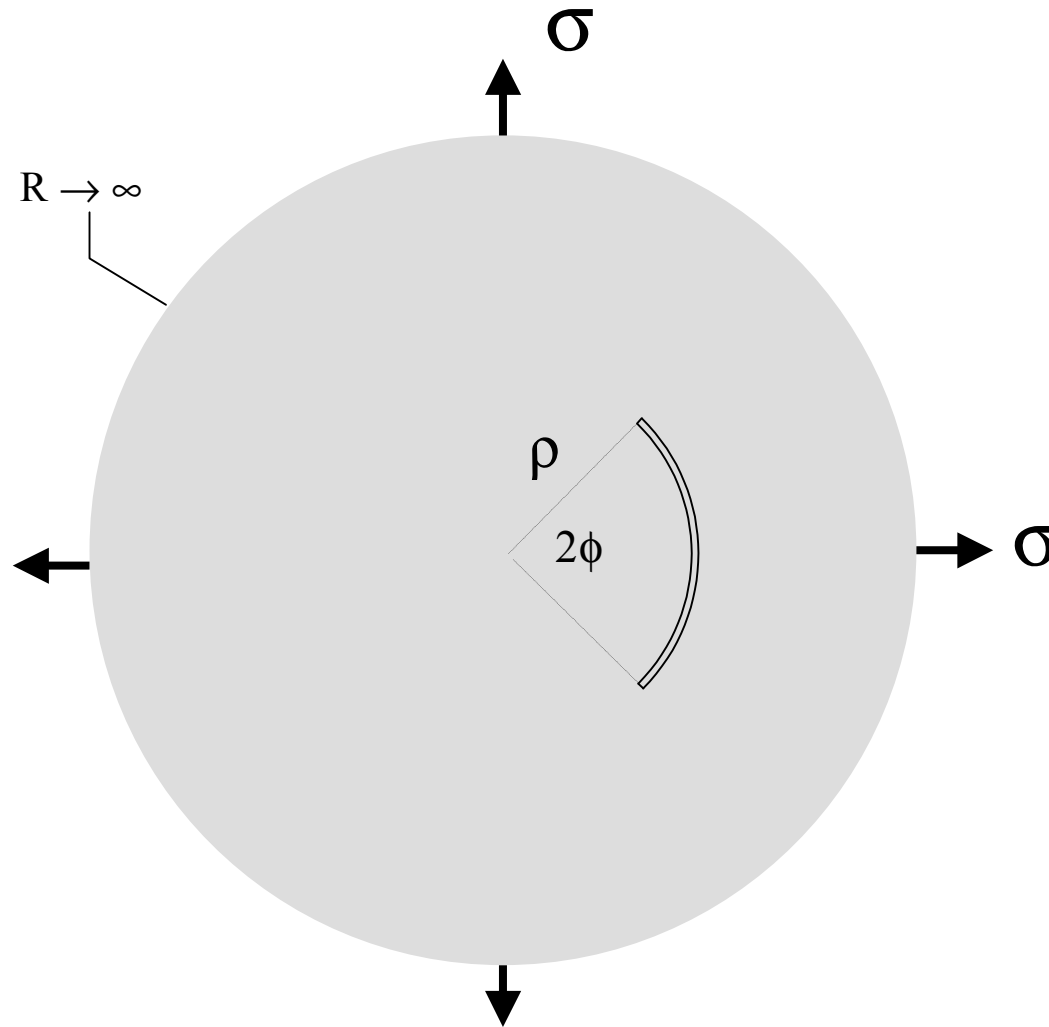
**For Configurational Equilibrium:**

$$\Delta\Pi = -\frac{\kappa + 1}{8\mu} \pi a^2 p^2 + (4\Gamma_0 - 2P)a + \frac{\kappa - 1}{2\mu} (2\Sigma_0 - P)pa$$

$$\frac{\partial \Delta\Pi}{2\partial a} = 0 \Rightarrow K_1^2 + \frac{(2\Sigma_0 - P) 2(\kappa - 1)}{\sqrt{\pi a} (\kappa + 1)} K_1 - \left(1 - \frac{P}{2\Gamma_0}\right) K_{1C}^2 = 0$$

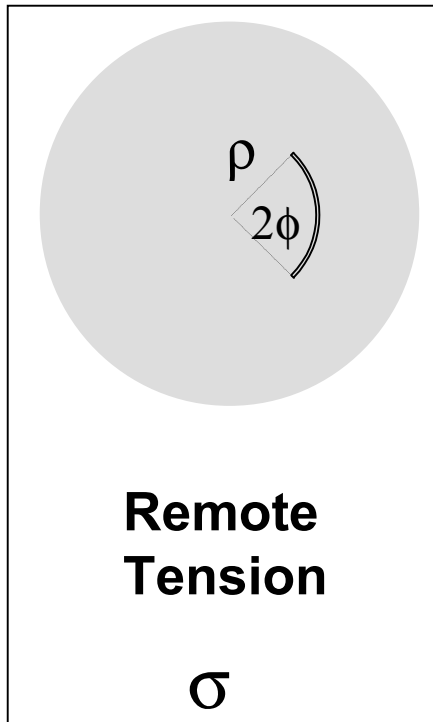
# Configurational Energy

## Circular-Arc Cracks, Remote Tension and Surface Stress



# Configurational Energy

## Circular-Arc Cracks, Remote Tension and Surface Stress



**Remote Tension:**

$$K^{(o)2} \equiv K_I^{(o)2} + K_{II}^{(o)2} = \sigma^2 \pi \rho \sin \phi / \left( 1 + \sin^2 \frac{\phi}{2} \right)^2$$

**Remote Tension and Surface Stress:**

$$K^2 \equiv K_I^2 + K_{II}^2 = \left( \sigma - \frac{\Sigma_0 \phi}{\pi \rho} \right)^2 \pi \rho \sin \phi / \left( 1 + \sin^2 \frac{\phi}{2} \right)^2$$

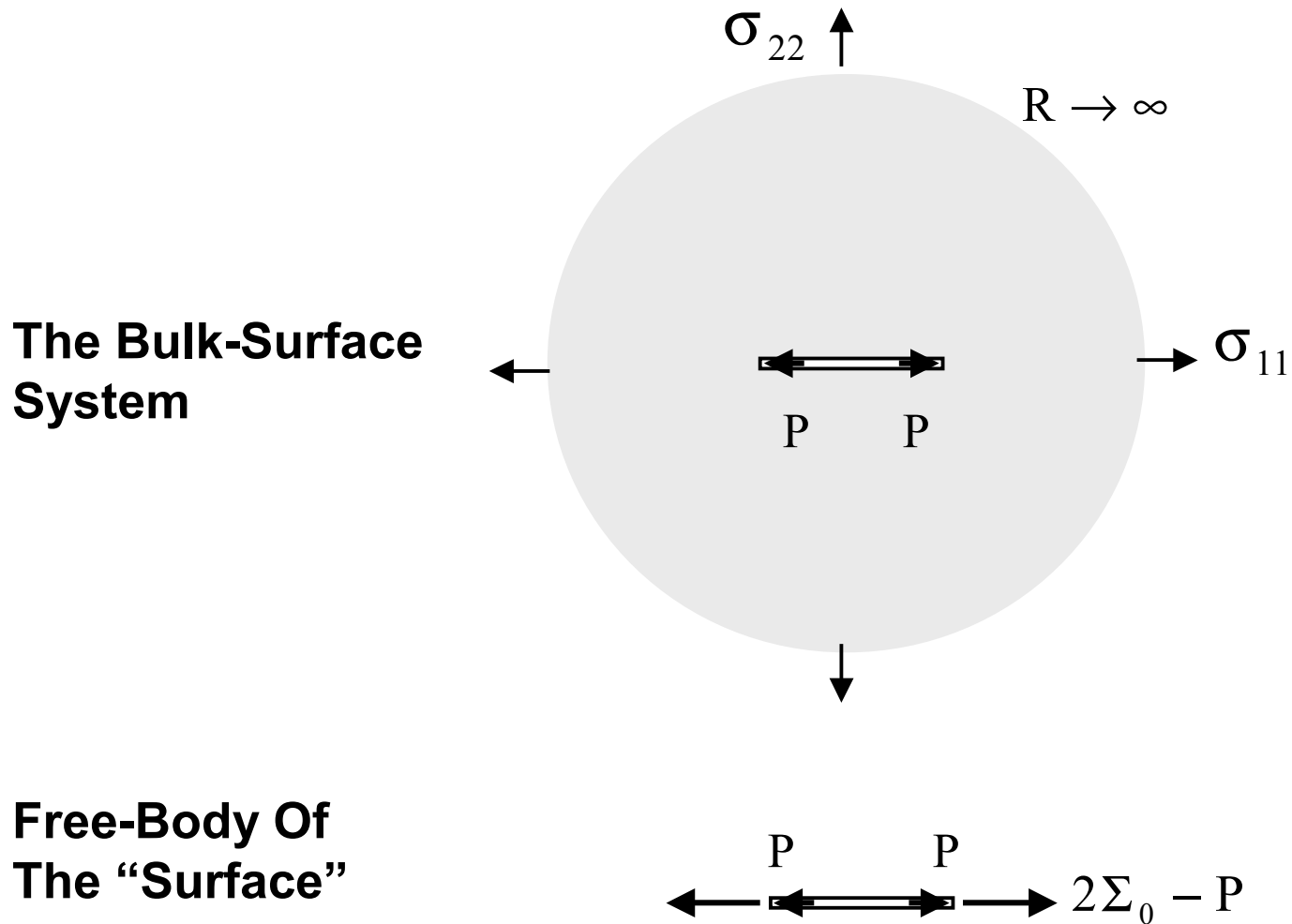
**For Configurational Equilibrium:**

$$\Delta \Pi = 4 \rho \phi \Gamma_0 - \frac{(\kappa + 1)(1 - \cos \phi)}{2\mu(3 - \cos \phi)} \left( \sqrt{\pi \sigma \rho} - \frac{\Sigma_0 \phi}{\sqrt{\pi}} \right)^2$$

$$\frac{\partial \Delta \Pi}{2 \rho \partial \phi} = 0 \quad \Rightarrow \quad K^2 - \left[ 4 \Sigma_0 \sin^2 \frac{\phi}{2} / \sqrt{\pi \rho \sin \phi} \right] K - K_{IC}^2 = 0$$

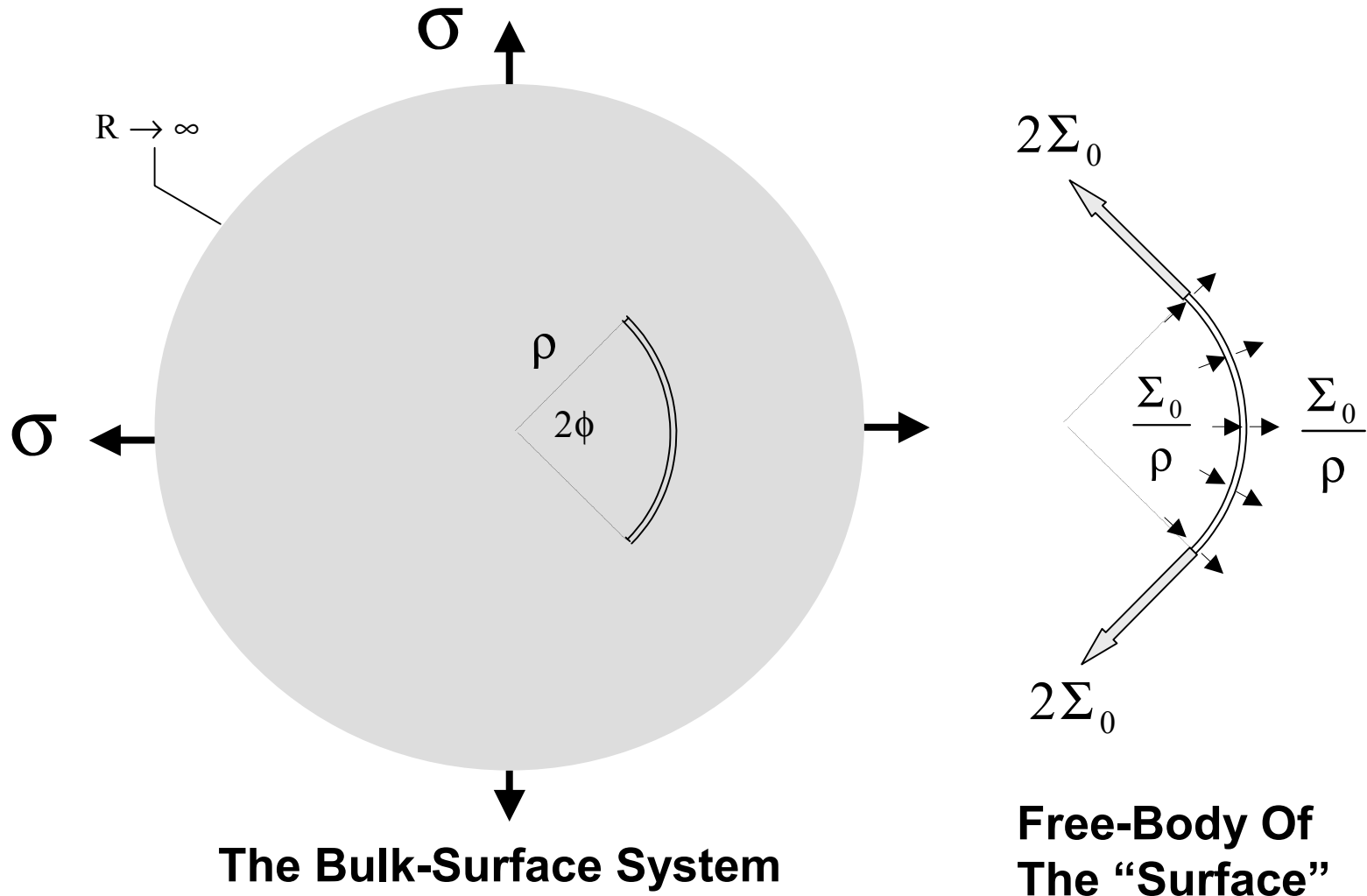
# Elasticity Solution

## Remote Loading, Surface Stress and Crack-Tip Load



# Elasticity Solution

## Circular-Arc Crack, Remote Loading and Surface Stress



# Elasticity Solution - Griffith Crack

## Remote Loading, Surface Stress and Crack-Tip Load

$$\left( u_{\alpha}, \varepsilon_{\alpha\beta}, \tau_{\alpha\beta} \right) = \left( u_{\alpha}^{(o)}, \varepsilon_{\alpha\beta}^{(o)}, \tau_{\alpha\beta}^{(o)} \right) + \left( u_{\alpha}^{(s)}, \varepsilon_{\alpha\beta}^{(s)}, \tau_{\alpha\beta}^{(s)} \right)$$

Total

Load

Surface Stress

### Strain Energies:

$$U_e \equiv \int W(\varepsilon_{\alpha\beta}) dA = \frac{R}{2} \int_0^{2\pi} [\sigma_{rr}(\theta) u_r(R, \theta) + \sigma_{r\theta}(\theta) u_{\theta}(R, \theta)] d\theta \\ + \frac{1}{2} (P - 2\Sigma_0) [u_1(a, 0) - u_1(-a, 0)]$$

$$U_e^{(o)} \equiv \int W(\varepsilon_{\alpha\beta}^{(o)}) dA = \frac{R}{2} \int_0^{2\pi} [\sigma_{rr}(\theta) u_r^{(o)}(R, \theta) + \sigma_{r\theta}(\theta) u_{\theta}^{(o)}(R, \theta)] d\theta$$

$$U_e^{(s)} \equiv \int W(\varepsilon_{\alpha\beta}^{(s)}) dA = \frac{1}{2} (P - 2\Sigma_0) [u_1^{(s)}(a, 0) - u_1^{(s)}(-a, 0)]$$

# Total Elastic Energy

$$\begin{aligned} U_e &= \frac{R}{2} \int_0^{2\pi} \left[ \sigma_{rr}(\theta) u_r^{(s)}(R, \theta) + \sigma_{r\theta}(\theta) u_\theta^{(s)}(R, \theta) \right] d\theta \\ &\quad + \frac{1}{2} (P - 2\Sigma_0) \left[ u_1^{(o)}(a, 0) - u_1^{(o)}(-a, 0) \right] \\ &\quad + U_e^{(o)} + U_e^{(s)} \\ &= (P - 2\Sigma_0) \left[ u_1^{(o)}(a, 0) - u_1^{(o)}(-a, 0) \right] + U_e^{(o)} + U_e^{(s)} \end{aligned}$$

## **Total Elastic Strain Energy:**

$$U_e = U_e^{(o)} + U_e^{(s)} + U_e^{(os)}$$

## **Interaction Energy:**

$$U_e^{(os)} = (P - 2\Sigma_0) \left[ u_1^{(o)}(a, 0) - u_1^{(o)}(-a, 0) \right]$$

# Energy of the Bulk-Surface System

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**Elastic Energy of the Bulk:**  $U_e = U_e^{(o)} + U_e^{(s)} + U_e^{(os)}$

**Surface Energy:**

$$U_s = 2 \int_{-a}^{+a} \Gamma(\varepsilon) dx_1 = 4a\Gamma_0 + 2\Sigma_0 [u_1(a,0) - u_1(-a,0)]$$

**Potential of External Loads:**

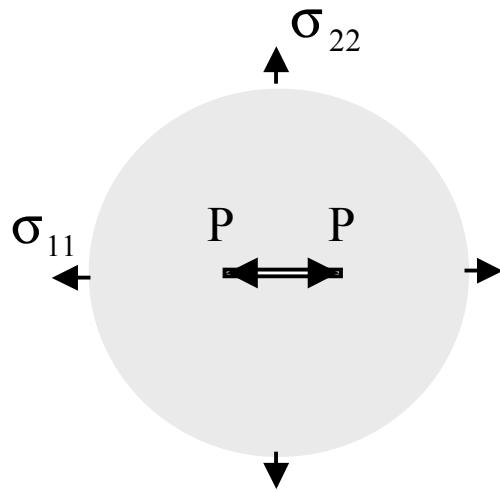
$$V = R \int_0^{2\pi} [\sigma_{rr}(\theta)u_r(R,\theta) + \sigma_{r\theta}(\theta)u_\theta(R,\theta)] d\theta \\ + P[2a + u_1(a,0) - u_1(-a,0)]$$

**Total Potential Energy:**

$$\Pi = U_s + U_e - V = -U_e^{(o)} - U_e^{(s)} - U_e^{(os)} + (4\Gamma_0 - 2P)a$$

## Final Results For $\sigma_{11}$ , $\sigma_{22}$ , $P$ , $\Sigma_0$ and $R \rightarrow \infty$

$$U_e^{(o)} = \frac{\pi R^2}{16\mu} \left[ (\kappa - 1)(\sigma_{11} + \sigma_{22})^2 + 2(\sigma_{11} - \sigma_{22})^2 \right] + \frac{\kappa + 1}{8\mu} \pi a^2 \sigma_{22}^2 + \dots$$



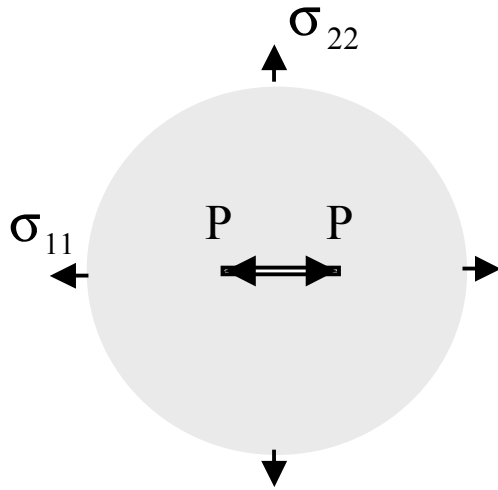
$$U_e^{(s)} = -\frac{(\kappa + 1)\Sigma_0^2}{2\pi\mu} \ln \left| \frac{z - a}{z + a} \right| \quad z \rightarrow a$$

$$U_e^{(os)} = \frac{(P - 2\Sigma_0)(\kappa + 1)a}{4\mu} (\sigma_{11} - \sigma_{22})$$

$$\Delta\Pi = -\frac{\kappa + 1}{8\mu} \pi a^2 \sigma_{22}^2 + (4\Gamma_0 - 2P)a - \frac{\kappa + 1}{8\mu} (4\Sigma_0 - 2P)(\sigma_{22} - \sigma_{11})a$$

$$\frac{\partial \Delta\Pi}{2\partial a} = 0 \Rightarrow K_1^2 + \frac{2\Sigma_0 - P}{\sqrt{\pi a}} \left( 1 - \frac{\sigma_{11}}{\sigma_{22}} \right) K_1 - \left( 1 - \frac{P}{2\Gamma_0} \right) K_{1C}^2 = 0$$

# The Role of P

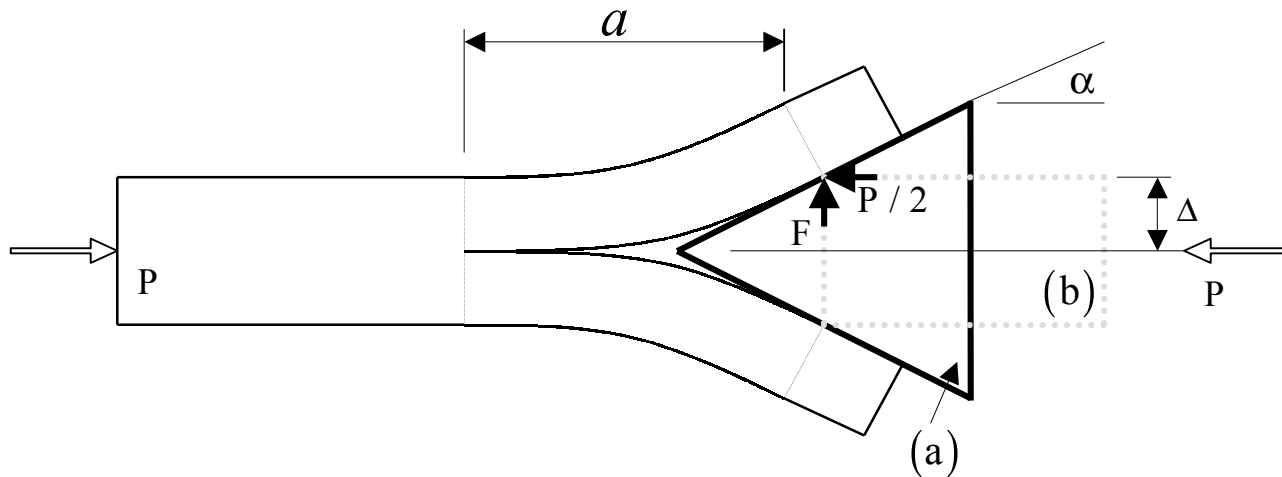


Let  $\sigma_{11} = \sigma_{22} = 0$

Then  $\Delta\Pi = (4\Gamma_0 - 2P)a$  and

$$\frac{\partial\Delta\Pi}{2\partial a} = 0 \Rightarrow P = 2\Gamma_0$$

# Splitting A Double-Cantilever Beam



**(a) A Smooth Wedge of Angle  $2\alpha$**

$$\Pi_{\alpha} = 2 \left[ \frac{2EI\alpha^2}{3a} + \Gamma_0 a \right], \quad \frac{\partial \Pi_{\alpha}}{\partial a} = 0 \quad \Rightarrow \quad P = 6\Gamma_0$$

**(b) A Smooth Edge of Thickness  $2\Delta$**

$$\Pi_{\Delta} = 2 \left[ \frac{3EI\Delta^2}{2a^3} + \Gamma_0 a \right], \quad \frac{\partial \Pi_{\Delta}}{\partial a} = 0 \quad \Rightarrow \quad P = 2\Gamma_0$$